SSC-445

STRUCTURAL SURVIVABILITY OF MODERN LINERS



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OCTOBER 2005

STRUCTURAL SURVIVABILITY OF MODERN LINERS

Discussions from International Maritime Organization, Society of Naval Architects and Marine Engineers and Ship Structure Committee have drawn attention to the need to explore the survivability of modern cruise liners in the flooded condition after collision damage. The structural capability of these ships to withstand additional weight of flood water, after sustaining structural damage from a collision has not been extensively evaluated.

The report presents results of an analysis of the ultimate strength of a second and significantly larger passenger ship in a flooded and structurally damaged, heeled condition. The report further investigates into the structural survivability of modern liners after flooding of multiple adjacent main compartments. The analysis is carried out using standard structural analysis and ultimate limit state analysis tools.

Г. H. GILMOUR

Rear Admiral, U.S. Coast Guard Chairman, Ship Structure Committee

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CONVERSION FACTORS (Approximate conversions to metric measures)

To convert from	to	Function	Value
LENGTH			
inches	meters	divide	39.3701
inches	millimeters	multiply by	25.4000
feet	meters	divide by	3.2808
VOLUME			
cubic feet	cubic meters	divide by	35.3149
cubic inches	cubic meters	divide by	61,024
SECTION MODULUS			
inches ² feet ²	centimeters ² meters ²	multiply by	1.9665
inches ² feet ²	centimeters ³	multiply by	196.6448
inches ⁴	centimeters ³	multiply by	16.3871
MOMENT OF INERTIA			
inches ² feet ²	centimeters ² meters	divide by	1.6684
inches ² feet ²	centimeters ⁴	multiply by	5993.73
inches ⁴	centimeters ⁴	multiply by	41.623
FORCE OR MASS			
long tons	tonne	multiply by	1.0160
long tons	kilograms	multiply by	1016.047
pounds	tonnes	divide by	2204.62
pounds	kilograms	divide by	2.2046
pounds	Newtons	multiply by	4.4482
PRESSURE OR STRESS	2		
pounds/inch ²	Newtons/meter ² (Pascals)	multiply by	6894.757
kilo pounds/inch ²	mega Newtons/meter ²	multiply by	6.8947
	(mega Pascals)		
BENDING OR TORQUE			
foot tons	meter tons	divide by	3.2291
foot pounds	kilogram meters	divide by	7.23285
foot pounds	Newton meters	multiply by	1.35582
ENERGY			
foot pounds	Joules	multiply by	1.355826
STRESS INTENSITY	2/2		
kilo pound/inch ² inch ⁷² (ksi√in)	mega Newton MNm ^{3/2}	multiply by	1.0998
J-INTEGRAL	2		
kilo pound/inch	Joules/mm ²	multiply by	0.1753
kilo pound/inch	kilo Joules/m ²	multiply by	175.3

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1 Executive Summary

This report is the continuation of Ref. 1, "Structural Survivability of a Modern Passenger Ship", where initial studies were done to investigate the ultimate strength of modern passenger ships. The report presents results of an analysis of the ultimate strength of a second and significantly larger passenger ship in a flooded and structurally damaged, heeled condition. Additional factors that may contribute to the ultimate strength of passenger ships are also studied.

The analysis is comprised of the following parts:

- Definition of three typical loading conditions for the ship.
- Determination of three different flooding scenarios that produce different levels of heel angles and structural loadings of the ship in calm water.
- Determination of a realistic wave loading for the flooded scenario.
- Evaluation of the ultimate or collapse strength of the ship in both the intact and damaged, heeled conditions.
- Comparison of the resulting loading to the strength capabilities of the ship.
- Comment on additional local factors that can affect the ultimate strength of the structure

The results of this analysis indicate that the ultimate strength of the hull girder is not severely affected by heel angles up to 15 degrees. For the studied cases, the ultimate strength is reduced by approximately 10% compared to the upright condition of the ship.

As found before, extensive flooding in the middle of the ship can place the ship in a sagged condition in still water where the upper decks are subject to compression loading.

When combined with a realistic and moderate wave loading, the ultimate strength of this ship, even after being subjected to a severe structural damage, should be adequate to withstand the loading from the worst-case flooding.

Additional factors can also affect the ultimate strength of the ship, but many variables affect the severity of these factors. With the reported margin in ultimate strength it is believed that these factors will not affect the survivability of the ship.

2 Introduction and Background

This report describes the continuing work from SSC Project SR-1439, reported upon in HEC Report 2003-24-01, (Ref. 1). It presents a further investigation into the structural survivability of modern liners after flooding of multiple adjacent main compartments. The analysis is carried out using standard structural analysis and ultimate limit state analysis tools.

In addition to the work presented in Ref. 1, these more detailed studies are done to assess factors not accounted for in the previous study. The structural capacity of the damaged ship is evaluated with the ship in a heeled condition, accounting for factors such as horizontal bending moments and shear forces. Other additional tasks include providing commentary on the influence of lateral pressure, initial deflections, welding induced residual stresses and end conditions as well as studies of the transverse bulkheads of a damaged ship.

Discussions held at IMO's Maritime Safety Committee, within various technical and research panels of SNAME, and at the Ship Structure Committee have drawn attention to the need to explore the survivability of modern cruise liners in the flooded condition after collision damage. In recent years modern cruise ships have grown in size to accommodate large numbers of passengers and crew. Measuring as much as 140,000 gross tons, these ships can carry as many as 5,000 persons.

Historical casualties have shown that a passenger ship may survive a flooding from a stability perspective but fail structurally. It is understood that passenger ships generally operate with a hogging still water bending moment that places the strength deck and other upper superstructure decks in tension and the bottom plating in compression. In the case of extensive flooding amidship, the additional weight of flood water potentially could substantially alter the bending moment and place the strength deck in compression. The structural capability of these ships to withstand this type of loading, after sustaining structural damage from a collision has not been extensively evaluated.

Herbert Engineering Corp. (HEC) has developed the HECSALV suite of naval architectural programs for the evaluation of ship casualties, salvage, and emergency response. The software is currently used by the casualty response departments of many of the major Classification Societies, by most major oil company fleets, by several navies, coast guards, and commercial salvors. HECSALV includes static intact and damage stability analysis, longitudinal bending analysis (in still water and superimposed on a static wave), and stress analysis based on specific intact or damage structural sections. The casualty analysis module of HECSALV has been extended to include pre- and post-processing of analysis from two US Navy applications, ULSTR and SMP, which are used in this investigation and are described in greater detail in Sections 4 and 5 of this report. The direct incorporation, model translation, and the pre/post-processing capabilities of ULSTR and SMP combined with the traditional naval architectural and casualty analysis capabilities of HECSALV make this the ideal tool to economically and efficiently meet the objectives of this SSC project.

3 Sample Ship

General Description

All of the analyses in this report are based on an un-built design of a modern cruise liner. The data for this ship such as hull and compartment geometry, lightship weight distribution and structural section properties was received from the USCG and reviewed by HEC for this project. Permission to use the generic data, with details regarding the specific ship identification removed, has been given by the shipyard and potential owner/operator.

The main particulars of the ship are listed in the following Table.

LBP	242.28 m
Beam	36.00 m
Depth	30.00 m
Design Draft, keel	8.469 m
Number of Decks	15

Table 1 - Main Particulars

The ship's hull and watertight compartments up to Deck 8 are modeled. All other spaces above Deck 8 are not part of the model and are assumed to be non-buoyant and free flooding in the flooded condition, see Fig. 1.



Figure 1 – HECSALV Model

Structural Design

A typical midship section is studied. This midship section is generally applicable between Frame 100 and 180 (between 0.28·LBP and 0.51·LBP forward of AP). The midship section drawing (Ref. 2) was used as a starting point in order to benchmark the sectional properties of the developed section with the values provided by the shipyard. Drawings and the supplied FE model were studied to determine what openings were appropriate to include as structurally effective in this section. It was observed from the FE model that a large opening was present through the middle of the ship, from the inner bottom and all the way up to Deck 15. Also, the side shell structure had numerous openings that were deducted from the "full" midship section. This "effective" section was used for the analysis.

While the project scope originally called for an analysis of several sections along the length of the ship, the provided reference drawings did only cover an area of about 55 m around the midship region of the ship, from Frame 100 through 180. The analyzed "effective" section is the minimum section observed within this area, and can be seen as a conservative approach to analyzing the ultimate strength of the ship. The "effective" section is used for the analysis throughout this report.

The full and "effective" sections are presented in Figure 2 and Figure 3. The resulting section properties from the "effective" section are somewhat higher but roughly comparable to the properties provided by the shipyard data in Reference 2. A comparison of the main properties of these sections is provided in Table 2. Table 3 lists some of the key features of the "effective" section used for the analysis.



Figure 2 – Full Midship Section



Figure 3 - "Reduced" Midship Section with openings deducted

	Midship Section	"Reduced"	Full
	Ref. 12	section	Section
Inertia (m4)	766	806	1085
Neutral axis ABL (m)	13.84	13.82	16.09
SM at Deck 15 (m3)	27.91	29.32	42.65
SM at BL (m3)	55.37	58.28	66.85

Table 2 - Comparison of Section Properties

Frame spacing	2860 mm
Typical longitudinal spacing	700 mm
Deck plate thickness	5-8 mm
Deck stiffener scantling	100x8 BP

Table 3 - Key structural characteristics of upper decks of analyzed section

4 Loads in Intact Conditions

Intact Loading Conditions (Still Water)

Three sample intact loading conditions and three final stage flooding scenarios forming a matrix of 9 damage conditions are statically evaluated (in still water plus static hog and sag waves) to study a number of combinations of loading and damage scenarios.

The intact conditions were requested in the project scope and the first and second were directly recommended by SSC. The third intact condition was selected based on bending moment output for all the intact cases provided. The three intact conditions are:

- 1. Minimum Freeboard condition
- 2. Typical departure condition
- 3. Minimum still water hogging bending moment

Note that the "minimum hogging moment" is the closest the ship can get to a still water sagging moment and is effectively the "maximum sagging moment.

The specific tank loadings were included in the data supplied for this project and a summary of the conditions is presented in the following Table.

Load Case	Stab1 - Max Freeboard Condition	Stab3a - Typical Departure Condition	Stab1a - Minimum Hogging Bending Moment Condition
Draft at MS	8.46 m	7.88 m	8.28 m
Trim	0.21 m Aft	1.03 m Aft	0.96 m aft
GM	2.56 m	2.15 m	2.53 m

Table 4 - Intact loading condition summary

The resulting Still Water Bending Moment and Shear results for the Maximum freeboard Draft loading condition are shown in Figures 4 through 6, and are somewhat similar for all three intact loading conditions.



Figure 4 – Still Water Bending Moment (Min. Freeboard intact condition)



Figure 5 – Still Water Bending Moment (Typical departure intact condition)



Figure 6 – Still Water Bending Moment (Min. hogging BM intact condition)

Wave Loading

In addition to the still water bending moment, the flooded conditions were to be evaluated in a moderate seaway of 3.5m significant wave height. Since the simple bending moment analysis utilized in HECSALV is based on superposition on a static hog or sag wave, a seakeeping analysis was performed to determine the equivalent static wave load to apply.

The seakeeping analysis was performed using the Ship Motions Program (SMP) which was first developed in 1981 at the David W. Taylor Naval Ship Research and Development Center as a tool to predict the response of U.S. Navy's ships to regular and irregular waves. The code has been updated several times since 1981, see Refs. 3 & 4. The dynamic bending moment is calculated in head seas in a seaway of 3.5m significant wave height at zero forward ship speed. The dynamic bending moments are developed based on the Ochi method (Ref 5), where a response spectrum for the vertical bending transfer function from SMP is developed based on the exposure time and wave heights. The short-term most probable extreme bending moment in a twenty-four hour period was calculated for the one load case and applied throughout the analysis for all combinations of load and damage cases.

A 3.0 m static sag wave with length equal to shiplength and trough amidships was found to produce equivalent moments to the 24 hour exposure to the 3.5 m seastate. A comparison between the two bending moments along the length of the ship is presented in Figure 7. This equivalent 3.0 m static sag wave with length equal to shiplength and trough amidships was applied to the matrix of combinations of three intact conditions and three damage scenarios, each with hog and sag wave moments, resulting in the bending moments indicated in the following tables.

The total bending moment is a function of the weight of the ship, its buoyancy and their distribution. The only difference between a still water bending moment and a total bending moment including a wave component is the buoyancy caused by the wave and inertial effects of the lightship weight. Therefore, this wave component can be

assumed to be relatively similar for all the checked conditions. Different drafts and different mass distributions from the intact cases might cause some small variation in the dynamic wave bending moments, but checks of the different cases indicate that this variance is negligible. Also, a heeled ship might cause some differences here, but SMP can only be run for an upright case.

The short-term most probable extreme shear force in a twenty-four hour period was also calculated with the same methodology as for the bending moment. These values are shown in Figure 8 below. Unlike for the bending moment, no single static wave will produce a curve similar to this calculated curve, so no further comparisons are made here.



Figure 7 - Dynamic Moment in a 3.5 m Seaway and the Equivalent Static Wave for 24 hours of exposure



Figure 8 - Dynamic Shear Force (MT) in a 3.5 m Seaway for 24 hours of exposure

5 Loads in Flooded Conditions

Flooding Scenarios

Three different flooding scenarios were considered for this report based on the damage scenarios described in Ref. 6 and the specifications in the project scope. These damage cases are the following:

- 1. Two-compartment high extent damage in the midship region
- 2. Four-compartment raking damage
- 3. Three-compartment damage in the midship region

Case 1 and 2 are cases specifically used in Ref. 6 and case 3 is the standard SOLAS damage extended longitudinally to 3 compartments. See Figure 13 for an illustration of the damage extents.

All these damage cases result in high sagging loads, which is the primary concern for a cruise ship with the low buckling capacity of the upper decks. While end damages can result in relatively high hogging loads, Reference 1 indicates that the hogging loads are still much lower than the ultimate capacity of the structure in hog.

While the damage extents describe what parts of the ship are damaged, the flooded areas are much larger due to progressive downflooding of compartments. These areas are therefore defined as flooded in HECSALV. The flooding extents are described in the following and the flooded compartments as defined in HECSALV are shown in Figure 9 through Figure 11.

High extent two-compartment damage

This damage is defined in Ref. 6 and extends from Deck 2 (5.8 m ABL) to Deck 6 (17.4 m ABL). The damage extends 8 m longitudinally centered on the bulkhead at Frame 148. The longitudinal extent makes this a two-compartment damage. Although the lower vertical extent is at Deck 2 with the deck itself not damaged, the flooding scenario in Ref. 6 describes downflooding on Deck 1. The effective damage in HECSALV will then include flooding compartments between Deck 1 and 2.



Figure 9 - Flooded compartments for High extent two-compartment damage

Raking Four-Compartment Damage

The four-compartment raking damage is a damage scenario that is expected to have a low probability of occurrence. The damage extends vertically 1 m around a base at 7.6 m above baseline and is on deck level 2 only. Longitudinally, the damage extends from aft of Frame 146 to fwd of Frame 198 with a total length of 36.6 m. Up-and downflooding points will effectively flood deck 1, 3 and 4 as well, so these compartments are also modeled as flooded in HECSALV.





Figure 10 - Flooded compartments for raking four-compartment damage

Three compartment damage

This is the standard transverse and vertical damage extent as defined by SOLAS extended longitudinally to involve flooding of 3 compartments. It includes damage over three compartments in the longitudinal direction, extending over the full depth of the ship, and 20% of the beam. The inner bottom area would be damaged if it resulted in a more severe flooding and damage stability effects, but for this case it does not and the inner bottom area is therefore left intact.



Figure 11 - Flooded compartments for three-compartment damage

Loading in the Flooded Condition

As specified in the SSC project description, the flooded conditions were to be evaluated in a moderate seaway of 3.5m significant wave height. The same 3m equivalent static wave as described previously is applied to all the damaged conditions resulting in the bending moments indicated in Table 5 through Table 7. Note that the reported bending moment values are the maximum value along the whole length of the ship. In some cases, the entire ship is in hog, even with a sag wave, and therefore, no value is indicated for the maximum sagging moment.

		Stab1 - Max Freeboard Condition						
Damage		4-comp F	Raking	High Extent 2-c	ompartment	Standard 3-compartment		
Wave		3m Hog	3m Sag	3m Hog	3m Sag	3m Hog	3m Sag	
Draft MS	m	10.949	10.566	8.571	8.855	10.113	9.938	
Static Heel	deg	16.8	7.2	19.8	3.4	15.8	5.1	
Trim at Perpendiculars	m	-0.67	-2.189	1.187	-0.981	0.693	-1.323	
GMt	m	2.454	1.043	-0.167	1.82	1.628	1.792	
Max Shear	MT	-3,408.64	-4,133.46	-3,928.34	1,889.52	3,371.95	3,447.30	
Max. Hog Bending Moment	MT-m	82,076H	29,651H	290,737H	138,511H	143,623H	41,242H	
Max. Sag Bending Moment	MT-m	33,738S	166,203S	-	-	-	84,006S	
Intact Weight	MT	53,400.89	53,400.89	53,400.89	53,400.89	53,400.89	53,400.89	
Flooding	MT	22,048.80	15,923.36	4,798.72	1,645.46	15,835.37	10,873.12	
Weight	MT	75,449.68	69,324.25	58,199.61	55,046.35	69,123.69	64,161.45	

Table 5	-Minimum	Freeboard	Condition.	Matrix of	f Damage an	d Wave Loadings
I upic c	17 I IIIIII WIII	1 I CCDOUI U	containing,	THULL IN O	Dunnage un	a mare houdings

			Stab3a - Typical Departure Condition				
Damage		4-comp	Raking	High Extent 2	-compartment	Standard 3-0	compartment
Wave		3m Hog	3m Sag	3m Hog	3m Sag	3m Hog	3m Sag
Draft MS	m				8.273		9.316
Static Heel	deg				4.1		6.5
Trim at Perpendiculars	m				-0.338		-0.702
GMt	m	No static	No static	No static	1.308	No static	1.231
Max Shear	MT	equilibrium	equilibrium	equilibrium	1,939.12	equilibrium	2,933.10
Max. Hog Bending Moment	MT-m	in still water	in etill weter		150,255H	(acc Teb 0)	48,275H
Max. Sag Bending Moment	MT-m	in still water	in still water	(see rab. 6)	-	(see rab. 8)	46,508S
Intact Weight	MT				49,250.65		49,250.65
Flooding	MT				1,296.08		9,955.13
Weight	MT				50,546.73		59,205.78

	Fable 6 -	Typical Departure	Condition, Matrix o	f Damage and	Wave Loadings
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		S	Stab1a - Minimum Hogging Bending Moment Condition					
Damage		4-comp	Raking	High Extent 2-0	compartment	Standard 3-cc	mpartment	
Wave		3m Hog	3m Sag	3m Hog	3m Sag	3m Hog	3m Sag	
Draft MS	m		10.376	8.361	8.693	9.871	9.76	
Static Heel	deg		8.1	21	3.5	16.9	5.2	
Trim at Perpendiculars	m		-1.598	1.891	-0.441	1.399	-0.741	
GMt	m	No static	0.845	-0.266	1.697	0.82	1.648	
Max Shear	MT	oquilibrium	-4,193.57	-3,654.24	1,793.22	3,285.04	3,432.67	
Max. Hog Bending Moment	MT-m	(ass Tab. 0)	26,820H	267,347H	121,948H	126,967H	33,285H	
Max. Sag Bending Moment	MT-m	(see Tab. 8)	176,300S	-	-	-	95,192S	
Intact Weight	MT		52,398.22	52,398.22	52,398.22	52,398.22	52,398.22	
Flooding	MT		15,568.85	4,773.98	1,563.91	15,313.32	10,511.86	
Weight	MT		67,967.07	57,172.20	53,962.13	67,711.55	62,910.08	

Table 7 - Minimum hogging bending moment condition, Matrix of Damage and Wave Loadings

Also note that in some cases, there is no static equilibrium for the ship in the damaged case with a wave included in the loads. For load case Stab3a, Typical Departure Condition, with a 4-compartment raking damage, there is no static equilibrium reached, even in still water, and no further analysis is performed. This is also reflected in Table 12 where the calculated bending moments are compared to the ultimate strength capabilities of the ship.

For the other cases where there is no static equilibrium with a hog wave applied, but where a static equilibrium is reached in still water and with the sag wave applied, more considerations need to be made. The fact that the ship is not stable with a 3-meter hog wave applied does not necessarily mean that the ship will capsize, and an approximation of the total load can be made with the bending moment from the still water case and the typical calculated wave bending moment presented in Figure 7.

While the bending moment for the undamaged ship is approximately 120,000 MT-m, it is smaller for the damaged cases since the damaged compartments give no contribution to the wave bending moment. The maximum bending moment contribution for the hog wave for the damaged cases is 63,00 MT-m. This value will therefore be used to check the cases where no static equilibrium is reached with a hog wave present, but where that static equilibrium is reached in still water and with a sag wave present. The maximum total bending moments for these cases are presented in the following table.

Load Case		Stab3a - Typical Departure Condition		Stab1a - Min Hog BM Condition
		High Extent 2-	Standard 3-	
Damage		compartment	compartment	4-comp Raking
Still water BM	MT-m	221,490	93,353	-110,187
Hogging wave BM	MT-m	63,000	63,000	63,000
Total approximated BM	MT-m	284,490	156,353	-47,187

Table 8 - Approximated hogging bending moment calculations

These moments are used for the comparison in Chapter 8.

Vertical and Horizontal Moments

For a ship with zero speed in still water there will be no horizontal bending moments acting in the "world" coordinate system, regardless of whether the ship is heeled or not. However, if the ship is heeled, there will be a horizontal bending moment present in the ship coordinate system equal to the calculated bending moment in the "world" coordinate system times the sine of the heel angle.

While HECSALV calculates a vertical bending moments in the "world" coordinate system, the effects of the horizontal bending moment in the ship coordinate system are effectively accounted for by calculating the strength of the ship around a heeled section. This is the case in both still water conditions and conditions with wave loads present.

A heeled section will cause two corners of the ship structure to be farther away from the neutral axis for the heeled condition, and therefore be more susceptible to buckling in the ultimate strength analysis. The full effect of this is discussed in the following section.

Note that a heeled ship with the same longitudinal weight distribution as a upright ship will give different bending moment results due to a change in longitudinal buoyancy distribution when the ship is heeled. This is the case for both still water conditions and conditions with wave loads present.

6 Ultimate Strength Analysis

Methodology

Ultimate strength was evaluated using the program ULTSTR. ULTSTR is a U.S. Navy developed program which evaluates the ultimate collapse strength and failure characteristics of ship structural sections. The components it uses to describe a structure are stiffened panels (gross panels) and hard corners. It provides a more sophisticated treatment of section failure than the simple correction factor applied to yield strength that is often used. Yielding, rupture (in tension), Euler beam-column buckling, and stiffener tripping are explicitly accounted for. Details of the analysis and file formats may be found in Refs. 7 and 8. The program also has the capabilities of analyzing a heeled section, and the final results incorporate an analysis of the sections at the flooded heel angle.

The ULTSTR Translator, which is incorporated as part of HECSALV, provides a means to perform ultimate strength analysis on structural sections defined in the HEC Section Modulus Editor. A model constructed with the Section Modulus Editor consists of a large number of independent parts. Stiffeners may be located next to plates, for instance, but no actual stiffened-plate relationship is described. Section Modulus models do not typically consist of components broken down into a form amenable to analysis by ULTSTR. The purpose of the translator is to determine the relationships between the structural pieces defined in the Section Modulus model, and to assemble them into the type of structural building blocks understood by ULTSTR.

The principal result of a typical analysis is the maximum moment which may be safely imposed upon the section. Details are also provided on the specific order of failure of structural elements and the failure mode they experienced.

All analysis is done on the "reduced" section as described in Chapter 3 of this report.

Intact Structure

The results of the analysis of the ultimate strength of the intact structure are presented in Figure 12 in the typical moment-curvature format. As expected, the curve is highly asymmetric, with the reduced moments in the sag direction reflecting the low buckling resistance of the structure of the higher decks.

The ultimate capacity of the structure in Hog is about 920,000 T-m. This point reflects the instant where all the bottom panels reach their tripping buckling capacity and collapse.

The ultimate capacity of the structure in Sag is about 360,000 T-m. This is the point where the upper superstructure starts collapsing, and it occurs well before any yielding is present in the structure, at approximately 34% of the upper flange yield point.

Also included in the figure are curves of the ultimate strength of the intact structure at different heel angles. The capacity of the structure is reduced with increased heel angles, but not dramatically. A 15 degree heel angle resulted in a reduced capacity of about 12% in Hog and 10% in Sag.

Also worth noting is that the ultimate strength curve is a little bit flatter at the failure point as a result of the fact that in the heeled condition the decks are at an angle compared to the bending plane of the structure and do not collapse simultaneously.



Figure 12 - Ultimate strengths of intact section at different heel angles

Damaged Structure

The analysis of the damaged structure consists of two parts in order to properly capture the envelope of the loading changing from an upright ship to a ship in a heeled position. The ultimate strength of the damaged section is calculated in both the upright condition and in the heeled condition as described in Table 9. The ultimate strength is then compared to the total bending moment for the undamaged ship in the upright condition, and to the total bending moment for the damaged, heeled ship in the heeled condition.



The results of the analysis for the upright condition of the damaged structure as presented in Figure 13, are presented in Figure 14. The 4-compartment raking damage is omitted from the analysis since the effective structural damage is very small. It is worth noting that the structural damage from the 2-compartment case also has very little influence on the calculated ultimate moment. The more significant structural damage in the 3-compartment case results in a significant reduction of the calculated ultimate moment.



Figure 14 - Ultimate strengths of damaged structure in upright condition

The results of the analysis of the same damaged structure for the maximum draft loading condition are presented in Figure 15. The different combinations of load and damage cases are analyzed at the equilibrium heel angles in still water as reported in HECSALV (see Table 9). The same tendencies as for the intact section are repeated here where the heel angle represents relatively small differences in the ultimate strength of the structure. Small damages at the lower parts of the hull structure also have little impact of the ultimate strength of the structure, whereas larger damage definitions where parts of the upper decks are damaged result in a significant reduction of the ultimate strength.

Equilibrium Heel Angles in Still Water	4-comp raking	High- extent 2- comp	Standard 3-comp
Stab1 - Max Freeboard Condition	11.5	10.3	10
Stab3a - Typical Departure Condition	-	14.9	16.1
Stab1a - Minimum Hogging Bending Moment Condition	13.4	10.9	10.4

Table 9	-	Heel	Angles
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2-comp - 3-comp - 4-comp - Intact-Even keel - Intact - 15 degrees heel

Figure 15 - Ultimate strengths of damaged structure in Max Freeboard Draft Condition

Effects of Damage Extents

While the previous chapter describes the ultimate strength for the damaged structure with effective structural damage extents equal to the damage assumed for the flooding, several factors might influence the effectiveness of the remaining structure. Damage to transverse structure can cause the remaining part of a damaged deck to have reduced effectiveness, or no effectiveness at all. One can easily imagine that damage to the transverse structure of a deck will result in double the effective length of the remaining stiffeners on the deck.

A study was therefore done on the effects of transverse damage extents and effectiveness of remaining deck structure on the 2-compartment and 3-compartment damages described earlier. The following cases are studied:

- 1. Damage of the side shell only (standard damage for the 2-compartment case)
- 2. Damage of the side shell with double effective length on the part of the decks outboard of the longitudinal girders on the damaged side to account for damaged transverse structure
- 3. Damage into longitudinal girders on damaged decks
- 4. Damage into longitudinal girders with double effective length on the rest of the decks on the damaged side to account for damaged transverse structure
- 5. Damage to include all of the damaged decks

In addition, the standard transverse damage extent for the 3-compartment case (20% of beam) was included in the analysis. All cases were run with no heel. The ultimate moments from these studies are presented in the following table.

High extent 2-comp					
	Hog	Sag			
Damage	Ultimate	Ultimate			
Extent	Moment	Moment			
1	918,000	360,000			
2	914,000	360,000			
3	914,000	359,000			
4	911,000	357,000			
5	907,000	357,000			

SOLAS 3-comp					
	Hog	Sag			
Damage	Ultimate	Ultimate			
Extent	Moment	Moment			
1	906,000	360,000			
2	901,000	333,000			
3	856,000	325,000			
4	845,000	237,000			
5	665,000	231,000			
20% of B	744,000	263,000			

Table 10 –Ultimate Moment Capabilities for varying damage extents

For the high-extent 2-compartment damage case, all the damaged structure is relatively close to the neutral axis of the analyzed section, so the changing transverse damage extents has little impact on the ultimate moment, in the order of maximum 1%.

The SOLAS 3-compartment damage case shows different results since the vertical damage extent extends all the way from the inner bottom and through the full height of the ship. An interesting effect is observed, where the big steps in ultimate moment happens at different combinations of damage/double effective stiffener length. For instance, between Damage Extent 1 and 2, there is little change in the hog ultimate moment since most of the decks are in tension. No structure is removed between these two cases, but the part of the decks outboard of the outboard longitudinal girders has an effective buckling length of double the normal web frame spacing. In the sag case most of these decks are in compression and there is a much more significant drop in ultimate moment.

The drop in ultimate moment in the hog case first happens when these parts of the decks are completely removed from the structure, as is the case for Damage extent 3. Note that doubling the effective length of certain parts of the decks are almost identical to removing the structure from the analyzed section for the sag case. This can be seen from the relatively small drop in ultimate moment in sag from Damage Extent 2 to Damage Extent 3.

Completely removing the decks on one side of the structure, with bottom and inner bottom intact, reduces the ultimate moment in both hog and sag by approximately 27%.

7 Additional Contributing Local Factors

One of the purposes of this report is to comment on the effects of some local factors to the ultimate strength of the ship. Several factors are investigated qualitatively and they are presented in the following.

Lateral pressure loads

Lateral pressure loads are usually not a factor in buckling of stiffeners in the longitudinal direction. The reason for this is that for most of the time, buckling of the stiffener will happen with deflections to alternating sides between the web frames, and the pressure will then alternate between constricting and aiding the buckling of the stiffener. For large lateral pressure loads a different buckling mode might occur (see the chapter End Connections below), and both these buckling modes are checked in ULTSTR. Also, typically for this ship the critical components which govern the ultimate strength in the sagging condition are the upper decks which are not subject to pressure loads.

Initial deflections

The effect of initial deflections on the buckling and ultimate strength of stiffeners and plating is a very complex problem to assess. Initial deflections can in fact both increase and reduce the buckling strength of structural members, all based on what shape the initial deflection has. A shape similar to the buckling mode of the structural member will reduce the buckling strength, while an initial deflection shape different than the buckling mode can increase the buckling strength. This is studied in detail in Ref. 9. For design purposes, a somewhat pessimistic assumption can be made with an initial deflection shape similar to the most common buckling mode of the structure.

Note that the reduction in ultimate strength of a plate based on an initial deflection shape is dependent on the socalled slenderness ratio of the plate, with more reduction of ultimate strength for more "slender plates". With the thin plating seen in the superstructure of passenger ships, this reduction can possibly be substantial.

Note that the ULTSTR program used for this analysis does not consider plate buckling a separate failure mode, but rather utilizes the reduced effective width of plating when stiffeners are buckling to account for plate buckling. For this analysis, the default initial deflections are assumed in ULTSTR with moderate to high distortions as defined in Faulkner's Expression. More details on how this is handled in ULTSTR can be found in Ref. 8.

Welding induced residual stress

In a ship structure, welding induced residual stresses will always be present. For a plate panel between stiffeners, parts of the plate will be in compression and parts of the plate will be in tension. A study on the effect of this was done in Ref. 9. It is noted that these residual stresses can significantly reduce the ultimate strength capacity of plate panels, but the magnitude of this reduction depends on several factors. First, the level of residual stresses will affect the reduction, but it is also noted that the reduction also depends on the assumed shape of the initial deflection of the plate panel.

ULTSTR has the possibility of defining residual stresses in two different ways; either by defining the residual compressive stress in the center of a plate panel as a factor of the yield stress, or by defining the extents of the tensile yield zone close to the stiffener as a multiplier of the plate thickness. For this analysis, the second option is used, with a tensile yield zone extent of 3 times the plate thickness on either side of the stiffener. The ULTSTR manual indicates that realistic values for the tensile yield zone can be in the range of 3 to 4.5 times the plate

thickness. A run with this value set to 5 gave about a 5% reduction in ultimate strength in sag, which is the loading condition of most interest here.

End Connections

The end connections of the stiffeners of the longitudinal structure can serve several purposes for the strength of the structure. First, any kind of flat bar or bracket will reduce the effective length of the stiffener and thus increase the buckling capacity of the stiffener. The effectiveness of any triangular or radiused brackets depends on the bracket size relative to the stiffener size; a small bracket with a large stiffener will have little or no effect and vice versa.

Secondly, any brackets or flat bars will help transfer shear load into the web plating at the ends of the stiffener. With no brackets or flat bars at the ends, this entire load will have to be transferred through the weld between the stiffener and the web plate. This is not specifically considered in this study.

Thirdly, any soft toes of the brackets will increase the fatigue life of the stiffener end connection. In an ultimate strength analysis this will have no effect on the results.

In addition to all this, the assumptions on whether the stiffener ends are considered fixed or pinned will have a great impact on the buckling strength. In fact, the behavior of the stiffener and the end connections is dependent on what kinds of loads are present. The most common buckling mode would be with the stiffener buckling to alternate sides between the web frames. The stiffener will then behave as if it were pinned at the ends, and this is the normal assumption for stiffener buckling calculations. Another possibility is that the stiffener always buckles to one side with effective clamped end connections. This failure mode is more of a plastic strength limitation than a buckling strength limitation and would probably not happen unless there is a large pressure head on one side of the plating. In any case ULTSTR checks both these failure modes.

Shear Forces

For a ship in an even keel intact condition, the shear forces will be vertical with most of these being resisted by the vertical plating of the ship such as side shell and longitudinal bulkheads. For passenger vessels, damage to the ship structure tends to increase these shear forces when compartments are flooded, and with damage to the structure, the shear capacity of the structure can also be reduced, with increased shear stresses in the remaining parts of the structure. This can lead to reduced buckling capacity of the plating of the side shell and longitudinal bulkheads if an interaction formula is used for calculating this. Although the plating might buckle earlier than if the structure was intact, two factors greatly reduce the importance of this effect.

First, the side shell and longitudinal bulkheads which are highly loaded in shear are not a major component of the bending strength of the hull structure. Therefore, buckling of parts of the side shell will not significantly reduce the overall bending strength of the hull structure. Also, the distribution of shear stresses is such that the highest shear stresses will be seen close to the neutral axis of the ship where the buckling stresses are low.

Secondly, plate buckling is not accounted for directly in ULTSTR. Buckling of single plate panels between stiffeners does not necessarily mean that the surrounding structure will fail, but it will reduce the buckling stiffness of attached stiffeners. This reduced effective width is accounted for in ULTSTR and therefore, plate buckling is incorporated in the evaluation of the ultimate strength of the structure.

A quick estimate of the whether the shear capacity of the hull is exceeded in any of the damaged cases studied here can be done by doing some simple but reasonable assumptions. The allowable positive still water shear force is approximately 3700 MT, and the ABS calculated positive wave shear force is approximately 3500 MT. It should therefore be safe to assume that the ship is designed to be able to take a total shear force of 7200 MT. The damage case mostly affecting the shear capacity of the structure, the SOLAS 3-compartment damage, has a reported worst shear force of 3500 MT, including the wave component from the assumed 3 meter wave. If a somewhat pessimistic assumption is made where only half the shear area is left after damage, the capacity could be approximately 3600 MT, and still higher than the reported shear force. This generally indicates that the shear

capacity of the ship is adequate, even after damage and flooding. More studies could be done where shear forces and capacities were studied in more detail along the length of the ship to obtain more accurate results.

8 Results – Loads vs. capabilities

Table 11 and Table 12 give summaries of the bending moment load versus the capabilities for the different flooded conditions. Table 11 represents the cases immediately after damage occurs, with damaged structure but with no flooding present. Table 12 represents the cases when the ship is flooded, and has reached static equilibrium in a heeled position.

Load Case	Damage		4-comp F	Raking	High Extent 2-	compartment	Standard 3-co	mpartment
	Wave		3m Hog	3m Sag	3m Hog	3m Sag	3m Hog	3m Sag
Stab1 - Max Freeboard	Max. Bending Moment	MT-m	415,812H	-	415,812H	-	415,812H	-
Condition	Ultimate strength	MT-m	920,000	360,000	918,000	360,000	744,000	263,000
Stab3a - Typical Departure	Max. Bending Moment	MT-m	423,192H	-	423,192H	-	423,192H	-
Condition	Ultimate strength	MT-m	920,000	360,000	918,000	360,000	744,000	263,000
Stab1a - Minimum Hogging	Max. Bending Moment	MT-m	395,982H	-	395,982H	-	395,982H	-
Bending Moment Condition	Ultimate strength	MT-m	920,000	360,000	918,000	360,000	744,000	263,000

Table 11 -	- Bending Mon	ent Loads versu	s Capabilities S	Summary for da	maged, upright	conditions

Load Case	Damage		4-comp	Raking	High Extent 2-	compartment	Standard 3-co	mpartment
	Wave		3m Hog	3m Sag	3m Hog	3m Sag	3m Hog	3m Sag
Stab1 - Max Freeboard	Max. Bending Moment	MT-m	82,076H	166,203S	290,737H	-	143,623H	84,006S
Condition	Ultimate strength	MT-m	839,000	331,000	844,000	331,000	664,000	252,000
Stab3a - Typical Departure	Max. Bending Moment	MT-m	No static	No static	284,490H	-	156,353H	46,508S
Condition	Ultimate strength	MT-m	equilibrium	equilibrium	814,000	314,000	620,000	260,000
Stab1a - Minimum Hogging	Max. Bending Moment	MT-m	-	176,300S	267,347H	-	126,967H	95,192S
Bending Moment Condition	Ultimate strength	MT-m	824,000	327,000	841,000	329,000	662,000	251,000

Table 12 - Bending Moment Loads versus Capabilities Summary for damaged and flooded conditions

Note that in one case, there is no static equilibrium for the combination of loading condition and damage case. In this case there is no need to evaluate the ultimate strength in this condition. Nor is it possible since the ship will capsize prior to reaching flooding equilibrium, even in calm seas. The cases highlighted in red indicate that the total bending moment is calculated by approximation as described in Chapter 5, since the ship is not in static equilibrium on the hog wave.

The reported bending moment values are the maximum value along the whole length of the ship. In some cases, the entire ship is in hog, even with a sag wave, and therefore, no value is indicated for the maximum sagging moment.

For all cases, the load never exceeds the capabilities of the structure, even in a heeled, damaged condition. The worst hog case is at the time of immediate damage, in the typical departure condition with a 3-compartment damage, where the hogging moment is 57% of the ultimate moment. The worst sag case occurs after flooding, in the minimum hogging bending moment condition with a 4-compartment raking damage, where the sagging moment is 54% of the ultimate moment.

9 Discussion of Results

This study demonstrates that the structural capabilities of a large modern passenger liner can generally withstand the loading of a heeled, damaged ship in a moderate seaway.

A matrix of three loading conditions and three damage scenarios is studied, and except for the cases where the ship does not survive from a flooding or stability standpoint, the total bending moments are well within the structural capabilities, even for the physically damaged structure. It should be noted that when the heeled, damaged ship is subject to head seas, the heel angle will vary with the wave position if asymmetrically flooded, and due to variation in wave direction in short-crested seas. All the analysis here is done at the static heel angle. The minimum reported ultimate strength in Sag, including both physical damage and heel, is approximately 250,000 MT-m. The highest combined total dynamic sagging bending moment is approximately 166,000MT-m or 67% of the ships ultimate strength capability. The minimum reported ultimate strength in Hog, including both physical damage and heel, is approximately 620,000 MT-m. The highest combined total dynamic sagging bending moment is approximately 166,000MT-m or 67% of the ships ultimate strength capability. The minimum reported ultimate strength in Hog, including both physical damage and heel, is approximately 620,000 MT-m. The highest combined total dynamic hogging bending moment is approximately 290,000MT-m or 47% of the ships ultimate strength capability.

Additional factors such as lateral pressure, initial deflections, weld induced residual stresses, end connections and shear forces are commented on and can all reduce the ultimate strength of the hull structure to a limited extent. However, it is not likely that these additional factors will affect the overall capability of the hull structure to withstand the anticipated loads.

10 References

- Structural Survivability of a Modern Passenger Ship, SSC Project SR-1439, HEC Doc. No. 2003-24-01, August 4, 2003
- 2. Principal Cross Section, Drawing, File Date March 4, 2004
- 3. "Users Manual for the Standard Ship Motion Program, SMP", W. G. Meyers, T. R. Applebee, and A.E. Baitis, David W. Taylor Naval Ship Research and Development Center, Ship Performance Department, September 1981, DTNSRDC/SPD-0936-01.
- 4. "SMP84: Improvements to Capability and Prediction Accuracy of the Standard Ship Motion Program SMP81", William G. Meyers and A. Erich Baitis, David W. Taylor Naval Ship Research and Development Center, Ship Performance Department, September 1985, DTNSRDC/SPD-0936-04.
- 5. "Wave Statistics for the Design of Ships and Ocean Structures", SNAME Transactions, Vol. 86, 1978, by M. K. Ochi.
- 6. Large Passenger Ship Safety, Time-to-flood simulations for a large passenger ship initial study, June 5, 2003
- 7. Adamchak, John C., An Approximate Method For Estimating the Collapse of a Ship's Hull in Preliminary Design, SNAME Proceedings, Extreme Load Symposium, 1984.
- 8. Adamchak, John C., ULTSTR(1996): The Revised Program for Estimating the Collapse of Ship Hulls or Hull Components Under Longitudinal Bending or Axial Compression.
- Effect of Initial Deflection Shape on the Ultimate Strength Behavior of Welded Steel Plates Under Biaxial Compressive Loads, Journal of Ship Research, Vol. 48, No. 1, March 2004
- 10. HARDER WP3, "Wave Height Distributions According to Damage Statistics", Report 3-00-W-2001-01-0, November 29, 2001.
- 11. SLF 45/3/5. "Investigations and proposed formulations for the factors "p", "r", and "v"; the probability of damage to a particular compartment or compartments, Report for the research project HARDER", 19 April 2002.
- 12. Paik, Wang, Kim, and Thayanballi, "Ultimate Limit State Design of Ship Hulls" SNAME Transactions, 2002.

Appendix A - Load Condition Details





Figure 16 - Minimum Freeboard condition, 4-compartment raking damage, Hog wave



Shear and Moment

Figure 17 - Minimum Freeboard condition, 4-compartment raking damage, Sag wave





Figure 18 - Minimum Freeboard condition, 2-compartment damage, Hog wave



Shear and Moment

Figure 19 - Minimum Freeboard condition, 2-compartment damage, Sag wave





Figure 20 - Minimum Freeboard condition, 3-compartment damage, Hog wave



Shear and Moment

Figure 21 - Minimum Freeboard condition, 3-compartment damage, Sag wave













Shear and Moment

Figure 24 – Minimum hogging moment condition, 4-compartment raking damage, Sag wave



Figure 25 – Minimum hogging moment condition, 2-compartment damage, Hog wave



Shear and Moment

Figure 26 – Minimum hogging moment condition, 2-compartment damage, Sag wave





Figure 27 – Minimum hogging moment condition, 3-compartment damage, Hog wave



Shear and Moment

Figure 28 – Minimum hogging moment condition, 3-compartment damage, Sag wave

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The following persons were members of the committee that represented the Ship Structure Committee to the Contractor as resident subject matter experts. As such they performed technical review of the initial proposals to select the contractor, advised the contractor in cognizant matters pertaining to the contract of which the agencies were aware, performed technical review of the work in progress and edited the final report.

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