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ABSTRACT

A 56 years old ship, K-maru, was investigated for items being concerned with her strength. Items investigated were the thickness of shell and deck plates, local and overall deformation of the hull, the wear of rivet heads and mechanical and other properties of steel plates of decks and hull. The characteristics of old plates investigated were chemical compositions, static and dynamic tensile strength, fracture toughness and fatigue strength. Thickness distribution of old plates with worn out holes due to corrosion were also measured.

Based on the results of investigations, the seaworthiness of K-maru was discussed. Results obtained were as follows:
1. The plate thickness reduction rate by corrosion of decks and shell plates at midship was about 5% in spite of her old age, though there existed some deep corroded parts in them.
2. The 5% of the whole hull deterioration rate corresponded to about 0.6% corrosion hole area ratio of the whole hull.
3. A small amount of deformation of the ship hull as a whole was responsible for the long term service and repeated repairs by welding.
4. The original old steel plates of 56 years showed good tensile strength, but poor weldability and fracture toughness.
5. Very rough surfaces of steel plates due to corrosion largely diminished their fatigue lives, because of decrease in thickness and stress concentration by surface roughness.
6. The seaworthiness level of the ship was very low. This level may reveal itself at the time of emergency.

BACKGROUND

The safe navigation characteristics of a steel structure sailing ship K-maru, 56-year-old at the time, was investigated. The principal dimensions of the ship were as following.

\[ L_{oa} \times B \times D \times d = 97.00 \text{ m} \times 12.95 \text{ m} \times 7.85 \text{ m} \times 6.15 \text{ m} \]

Gross tonnage = 2244.64 ton.

K-maru was constructed by using rivets, and so the original steel plates used were those without considerations of their weldability.

Figure 1 shows the short history of the navigation of K-maru. Annual navigation mileage is 20000 - 25000 nautical miles. And in this mileage, 10000 miles are performed by sailing (shown by solid circles in the figure). In Fig. 2, shaded plates show the side and bottom plates replaced by new ones in the way of her history. Numbers of replaced plates are 40 for port side and 55 for starboard side. In Fig. 3, replacements of deck plates are shown. The first replacement took place at the time when K-maru was 47-year-old. Whole plates of long poop deck and 80% of upper deck were replaced, and no plates of No. 2 and No. 3 decks were replaced at the time of investigation.

These replacements of decks, side and bottom plates were performed because of excessive local deterioration of plates due to corrosion. We investigated into the strength of K-maru and discussed the safety navigation characteristics of her, because of the anticipation that deterioration might proceed in steel plates which were not yet precisely inspected, and that there might be troubles of welding due to connecting original steel plates and new ones (replacements of side and bottom plates were conducted by using rivets but those of decks and other members other than side and bottom plates were conducted by welding), and also because of the fact that many heads of rivets were severely worn out.

METHOD OF INVESTIGATION

Investigations were conducted at two times of docking of K-maru. Items of investigations were: amounts of reduction in thickness of plates at midship due to corrosion, overall and local deformations of decks and hull and reduction in volume of rivets due to corrosion. Analysis of chemical compositions, precise measurement of thickness distribution and tests concerning strength were conducted for K-maru's old plates which were replaced by new weldable steel plates.

AMOUNT OF PLATE THICKNESS REDUCTION

Amounts of plate thickness reduction due to corrosion were measured at two points on every plate consisting hull structure at midship by using electromagnetic thickness meter. Table 1 shows the results. In the table, A6, B6 and so on are the names of plates shown in Fig. 2. The plate B8 starboard side was under construction of replacement at the time of investigation. Though there are plates like C8 port and H8 starboard which are corroded excessively, other plates are not so much. Mean values of thickness reduction of most plates are rather small, and the nominal total mean thickness reduction of plates at midship is nearly 5%, though there are plate like G8 and H8 port which have deep locally corroded parts.
### Table 1. Plate Thickness at Midship Section

<table>
<thead>
<tr>
<th>Plate No.</th>
<th>Port Side</th>
<th>Starboard Side</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>Age (mm)</td>
<td>th</td>
</tr>
<tr>
<td>Bottom</td>
<td></td>
<td></td>
</tr>
<tr>
<td>A6</td>
<td>14.2</td>
<td>55</td>
</tr>
<tr>
<td>B6</td>
<td>13.2</td>
<td>55</td>
</tr>
<tr>
<td>C6</td>
<td>13.2</td>
<td>55</td>
</tr>
<tr>
<td>D6</td>
<td>13.2</td>
<td>11-15</td>
</tr>
<tr>
<td>Bilge</td>
<td>E7</td>
<td>13.2</td>
</tr>
<tr>
<td></td>
<td>F1</td>
<td>12.2</td>
</tr>
<tr>
<td></td>
<td>G8</td>
<td>14.2</td>
</tr>
<tr>
<td></td>
<td>H14</td>
<td>14.2</td>
</tr>
<tr>
<td></td>
<td>I8</td>
<td>13.2</td>
</tr>
<tr>
<td></td>
<td>J7</td>
<td>12.2</td>
</tr>
<tr>
<td></td>
<td>K7</td>
<td>14.2</td>
</tr>
<tr>
<td></td>
<td>L8</td>
<td>13.7</td>
</tr>
</tbody>
</table>

Mean Value

<table>
<thead>
<tr>
<th>Name</th>
<th>Age (mm)</th>
<th>th</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>13.7</td>
<td>94 %</td>
</tr>
</tbody>
</table>

\( t_0 \): Original Thickness, \( t \): Thickness, *) local corrosion part

![Fig 2 Replacement of Side Shell Plate](image)

It seemed to be rather controversial that the fact mentioned above and the fact that there found many corrosion holes in deck plates. But this was because that corrosion holes or deep corroded parts were very much isolated and other parts were not so much affected by these.

In order to clarify the characteristics of corrosion hole, we measured the detailed thickness distribution of two old plates (a shell plate and an inner plate attached to a shell plate), both of which had corrosion holes. Tables 2 and 3 show the results of measurement. Plate thicknesses were measured by a large size micrometer at cross points of vertical and horizontal lines which were apart from each other by 20 mm. In the tables shaded parts have plate thickness less than 2 mm, and these parts are possible to be considered as parts of holes. Plate thickness of 2 mm corresponded to 17-20% of original one, and surfaces of such plates showed very rough appearance. And it was difficult to completely eliminate rust from their surfaces. The value 2 mm mentioned above had no theoretical meaning but just for practical purpose.

It is clear from these tables that the area of corrosion holes and directly surrounding excessively thin places are very much limited. Decrease in thickness of parts of the plate somewhat away from holes, is relatively small. There seems to exist an analogy between the phenomenon mentioned above and the fact that there are many corrosion holes (expressed by corrosion hole ratio \( V \)) in spite of rather small amount of mean nominal deterioration due to corrosion (expressed by whole area corrosion ratio \( \Delta \)) of 5% in Table 1. So we attempted to combine the two phenomenon in order to obtain the relation between \( V \) and \( \Delta \), by using results in Tables 2 and 3.
Table 2. Thickness Distribution of a Plate with Corrosion Holes
(Side Shell Plate K-13-P
Original Thickness = 10.16 mm)

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 | 13 | 14 | 15 | 16 | 17 | 18 | 19 | 20 | 21 | 22 | 23 | 24 | 25 | 26 | 27 | 28 | 29 |
| A | B | C | D | E | F | G | H | I | J | K | L | M | N | O | P | Q | R | S | T | U | V | W | X | Y | Z | AA | AB | AC | AD |

Table 3. Thickness Distribution of a Plate with Corrosion Holes
(Inner Plate of Side Shell Plate J-13-P
Original Thickness = 11.68 mm)

| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 | 13 | 14 | 15 | 16 | 17 | 18 | 19 | 20 | 21 | 22 | 23 | 24 | 25 | 26 | 27 | 28 |
| A | B | C | D | E | F | G | H | I | J | K | L | M | N | O | P | Q | R | S | T | U | V | W | X | Y | Z | AA | AB | AC | AD |

Figure 4 shows the relation between the corrosion ratio \( \delta \) and the area ratio \( A_r \) for plates K-13-P (Table 2) and inner plate of J-13-P (Table 3). Where \( A_r \) is the Ratio of \( A \) (an area of a portion of concern around corrosion hole or holes imaginarily taken concentrically of it or them), to the area of the whole plate \( A_w \). That is to say \( A_r = A / A_w \). The corrosion ratio \( \delta \) is the mean corrosion volume ratio calculated in the area of concern \( A \) stated above. As shown by both curves in Fig. 4, \( A_r \) is going to saturate as \( \delta \) is approaching to 100%.

Let us assume that the whole area corrosion rate \( \Delta \) is the value of \( \delta \) at \( A_r = 100\% \), and that \( V \) is the ratio of corrosion hole or holes area to the whole area of the plate. In Fig. 5, two points (vacant circles) show the calculated relation between \( \Delta \) and \( \delta \) using Tables 2 and 3. It is natural that \( \Delta = 0 \) at \( V = 0 \). Then we get the quadratic curve which passes three points mentioned above as following:

\[
V = 7.76 \times 10^3 \Delta^2 + 9.97 \times 10^2 \Delta
\]
We assume that the relation of equation (1) can be applied to the relation of Δ and V for whole ship. Putting A = 5.00 in equation (1), we get V = 0.693. The corrosion hole rate of 0.693% is very big value. It is because that if there are corrosion holes in the bottom plating it is very dangerous for the ship (K-maru had such experience in her life). It can be understood from this analysis that mean thickness reduction rate of 5% at midship (shown in Table 1) is not a trivial but a serious matter.

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The vertical deformation is dependent on positions of blocks.

Table 4. Deformation of Keel

<table>
<thead>
<tr>
<th>Station</th>
<th>Vertical Displacement</th>
<th>Transverse Displacement</th>
</tr>
</thead>
<tbody>
<tr>
<td>SS0</td>
<td>0.19</td>
<td>0.18</td>
</tr>
<tr>
<td>SS1</td>
<td>0.21</td>
<td>0.20</td>
</tr>
<tr>
<td>SS2</td>
<td>0.23</td>
<td>0.22</td>
</tr>
<tr>
<td>SS3</td>
<td>0.25</td>
<td>0.24</td>
</tr>
<tr>
<td>SS4</td>
<td>0.27</td>
<td>0.26</td>
</tr>
<tr>
<td>SS5</td>
<td>0.29</td>
<td>0.28</td>
</tr>
</tbody>
</table>

Fig. 6 shows horizontal deformations of side plates at the height from the keel line to be 4m (4WL) and 7m (7WL). In the figures, (+) indicates outward and (-) indicates inward deformations. At 4WL, there are little horizontal deformations around midship (at SS4 and SS5) and there are outward deformations at bow side and a little bit inward deformations at stern side. At 7WL, quite different from deformations at 4WL, whole deformations occur to port side direction. The maximum deformation at 4WL is 302 mm and this value corresponds to 2.3% of the breadth.

Deformation measurements of a bulkhead at Fr.60 (not far from midship = Fr.64 1/2) show that the maximum deformation is 26 mm which is equal to 0.20% of the ship breadth at the position. This value is negligible small from a view point of the reduction in buckling strength.

Figures 6 and 7 show horizontal deformations of side plates at the height from the keel line to be 4m (4WL) and 7m (7WL). In the figures, (+) indicates outward and (-) indicates inward deformations. At 4WL, there are little horizontal deformations around midship (at SS4 and SS5) and there are outward deformations at bow side and a little bit inward deformations at stern side. At 7WL, quite different from deformations at 4WL, whole deformations occur to port side direction. The maximum deformation at 4WL is 302 mm and this value corresponds to 2.3% of the breadth.

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DEFORMATION OF HULL

Measurements of ship hull deformation of K-maru were conducted at the time of her docking. Hull deformation was determined by measuring the distances between points on the hull and their projections on the plane containing a line which passed two end points of original straight keel line and four lines which extended horizontally from both side from previous two points. Transits, strings with weights, levels and tape measures were used for measurement.

Deformation measurements of bulkheads were conducted based on a horizontal base line connecting two points, each on a touching line of a bulkhead end to side shell.
Other than mentioned above, deflections of the bulkhead at Fr116 (not far from stem), vertical deflections of side lines of the upper deck and the overall deflections of the long poop deck were measured. In the measurements of the long poop deck, the maximum upward deflection was 89 mm, the maximum downward - 202 mm. And it was revealed that there were slow humps and hollows all over the decks.

It is considered that deformations measured are piled up due to so many times of long navigations, repairs of the rivet ship by welding, and unbalanced replacements of port and starboard sides shell plates. Especially the unbalanced replacements mentioned above are considered to be the reason of the deformation of the keel line. It is considered to be impossible to repair these deformations.

CHARACTERISTICS OF OLD PLATE

Side shell and deck plates of K-maru (here after called 'old plates'), which were replaced by new ones at the time of her repair docking, were carefully examined on such items as chemical compositions, static tensile strength, fracture toughness, Charpy impact strength and fatigue strength. Plate thickness and surface roughness measurements were also conducted on plates with excessive corrosion.

Chemical compositions

Table 5 shows chemical compositions of four old shell plates (named SS-1 through SS-4) with those of a commercial steel NK-KA. Old plates show that they have wide variations in an amount of carbon contents. Sulphur contents of them show the critical value or well over the value of the standard for NK-KA. Because of the fact that both carbon and sulphur contents of the plate SS-2 were very high, effects of these high contents on the properties of strength were investigated.

High contents of sulphur was also revealed by a so called sulphur print. Table 6 shows carbon equivalents C_{eq} of SS-2, by which weldability of the steel can be determined. C_{eq}’s of other steel plates with the NK-KA grade commercial steel of today are also calculated and shown in Table 6 and Fig. 6 (diagram to determine conditions of welding based on C_{eq} and plate thickness to be welded). C_{eq}’s are calculated by the following equation, neglecting the contents of chrome, molybdenum and vanadium.

\[ C_{eq} = C + \frac{Mn}{6} + \frac{Si}{24} \]  

(2)

Figure 8 indicates that the welding of SS-2 should be done by limited conditions.

Static tensile strength

Static tensile test were conducted for 17 pieces of test specimens of JIS 13A type (parallel part length = 120 mm, breadth = 20 mm, distance for elongation measurement = 80 mm, over all length = 340 mm). Five old plates (2 side shell plates and 3 deck plates) were used for test specimens. Surfaces of test specimens were as corroded or slightly shaved on an enormously corroded side for the convenience of testing. Test speed settled between chucks of testing machine was 1 mm/min. Test results for steel plates SS-2 and SS-3 are shown in Table 7. It should be noted that test specimens of SS-2 fractured by brittle mode at room temperature of 10°C. No clear yield points were shown at SS-2 specimens. Test specimens other than SS-2 fractured by ductile mode. For these specimens, the yield strength \( \sigma_y \) (calculated at minimum area section between two points for elongation measurement) was 23.1-24.8 kgf/mm² (226 - 234 MPa), and tensile strength \( \sigma_t \) (calculated same manner as \( \sigma_y \)), was 41.3-48.3 kgf/mm² (405 - 473 MPa). And these values were within the specified value of NK-KA (\( \sigma_y \geq 24 \text{ kgf/mm}^2 = 235 \text{ MPa} \), \( \sigma_t = 40 - 50 \text{ kgf/mm}^2 = 400 - 490 \text{ MPa} \)) as shown in Table 7.

Table 7. Mechanical Properties

<table>
<thead>
<tr>
<th>Material</th>
<th>Yield Strength ( \sigma_y ) (kgf/mm²)</th>
<th>Tensile Strength ( \sigma_t ) (kgf/mm²)</th>
<th>Elongation ( \varepsilon ) (%)</th>
<th>Note</th>
</tr>
</thead>
<tbody>
<tr>
<td>SS-2 TP1</td>
<td>—</td>
<td>52.7</td>
<td>12.2</td>
<td>Brittle</td>
</tr>
<tr>
<td>SS-2 TP2</td>
<td>—</td>
<td>84.4</td>
<td>13.8</td>
<td>Brittle</td>
</tr>
<tr>
<td>SS-3 TP1</td>
<td>27.5</td>
<td>45.4</td>
<td>29.3</td>
<td>Ductile</td>
</tr>
<tr>
<td>SS-3 TP2</td>
<td>26.2</td>
<td>44.0</td>
<td>34.1</td>
<td>Ductile</td>
</tr>
<tr>
<td>NK Standard</td>
<td>37.0</td>
<td>49.4</td>
<td>33.7*</td>
<td>As Rolled</td>
</tr>
</tbody>
</table>

*1 Distance Measured 200mm

Photographs 1 and 2 show SEM fractographies of SS-2 (brittle fracture) and SS-3 (ductile fracture) specimens. In Photo 1, a origin of brittle fracture is shown. The ductile fracture pattern shown at lower left hand side of the photo changes to brittle one at the origin. All fracture surface is covered by dimple pattern in case of Photo 2 (SS-3).

Fracture toughness test was conducted for the plate SS-2 which showed brittle mode at static tensile test at room temperature.

Dynamic tensile tests were conducted at strain rate of 0.09-0.13 /sec, at temperatures 0°C and 18±2°C. For SS-2 specimen,
almost all surface was observed as brittle mode at 0°C. At 20°C, almost all surface was observed as ductile mode and only limited parts showed crystalline fracture surface of brittle mode. Results of dynamic tensile tests almost corresponded to those of static tests, excepting that specimens with very rough surface due to corrosion fractured by ductile mode with relatively small value of elongation ε of 20% (mean value of ε for specimens by ductile mode was 31%, by brittle mode - 12%). It is considered that the effects of stress concentration due to corrosion revealed itself strongly at dynamic tests.

Fracture toughness tests were conducted for SS-2. As the original thickness of SS-2 was 0.52 inch (13.21 mm) and was heavily corroded, it was impossible to conduct the perfect fracture toughness test by CT test specimen. And so modified specimens shown in Fig. 9 were used.

Mechanical notch specimens with a tip radius of 0.2 mm were tested at 18°C, 0°C, -20°C, -40°C by testing speed of 20 mm/sec. One specimen, which had a notch size of 2.5 mm in depth with a fatigue crack, was tested at -40°C.

Figure 10 shows the results. The solid line in the figure shows the transition curve connecting lower bound of experimental data values of specimens with mechanical notches. The broken line in the figure corresponds to the transition curve for specimens with fatigue crack, being obtained by shifting the solid line to higher temperature side by 30°C [2]. In the case that an experimental value of $K_c$ is equal to $K_{lc}$, the thickness $t$ of the specimen satisfies the following equation.

\[ t = 2.2 \left( \frac{K_c}{\sigma_f} \right)^\frac{1}{2} \]  

\[ \text{Fig. 9 Test Specimen for CT Test} \]

\[ \text{Fig. 10 Temperature Dependency of Fracture Toughness } K_c \text{ of SS-2} \]
Assuming, in the equation (3), that \( t = 12 \text{ mm} \) and \( \sigma_y = 25.8 \text{ kgf/mm}^2 \), then we get the following result.

\[
K_c \leq 56.5 \text{ (kgf-mm)}^2
\]  

The region of \( K_c = K_c \) for wide range of temperature can be calculated as shown by the shaded area in Fig. 10, assuming temperature dependency of \( \sigma_y \) as equation (5) and conducting the same calculation as equation (4).

\[
\sigma_y = \sigma_{y_T} \exp \left[ \frac{(329.6 - 66.5 \ln \sigma_{y_T})}{T - 293} \right]
\]  

Where,

- \( \sigma_{y_T} \) = yield strength (kgf/m²) at temperature of \( T \), \( K \)
- \( \sigma_{y_T} \) = yield strength (kgf/m²) at temperature of 293 K.

The fracture toughness of SS-2 can be discussed based on the results obtained hitherto. By using the absorbed energy, \( E_r \) (kgf-m) of Charpy impact test at temperature \( T \), \( K_c \) can be calculated by using the equation by Rolfe et al. as following.

\[
K_c = 120 \left( \frac{E_r}{\sigma_y} \right)^{3/4}
\]  

\( K_c \) value in Fig. 10 can be transformed into \( K_c \) by using the following equation in consideration of plate thickness effects on \( K_c \).

\[
K_c = K_c / F(t)
\]

\( F(t) = 1 + 0.043 (t/40 - 1) \) \( (t = 40 \text{ mm}) \)

In Fig. 11, \( K_c \) value calculated by the equation (6) (curve I), \( K_c \) value by the equation (7) (curve III), \( K_c \) for commercial steel plate (curve II) and also \( K_c \) value obtained from CT test as shown by the broken line in Fig. 10 (curve IV), are shown.

Though the CT test shows no fracture toughness values (Kc) which satisfy \( K_c \) conditions, there is a possibility of a crack propagation under not far from \( K_c \) conditions, in case of crack existence in structural members with large structural constraints.

The stress intensity factor \( K_c \) in the case of a crack (2c in length) in infinite plate under uniform tensile stress \( \sigma \) as shown in Fig. 12 is expressed as follows.

\[
K_c = \sigma \sqrt{\pi c}
\]

For SS-2, the crack length, at which \( K_c \) in the equation (8) becomes \( K_c \) (brittle fracture may occur at this value) can be obtained by using curve I and III in Fig. 11. Results are shown in Table 9, assuming working stress \( \sigma \) to be 10 and 20 kgf/mm². Under condition of \( 10^\circ C \) and \( \sigma = 10 \text{ kgf/mm}^2 \), 2c is 92 mm or 44 mm. These values are less than 100 mm (considered to be the critical crack length to be found by naked eye inspection), and there are possibilities that such cracks exist in plates equivalent to SS-2 not found by inspection. Such a case is very dangerous for K-maru.

### Table 8. Charpy Impact Test Results

<table>
<thead>
<tr>
<th>Material</th>
<th>Chemical Compositions (%)</th>
<th>Impact Test Results</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>C</td>
<td>S</td>
</tr>
<tr>
<td>NK-A Grade Commercial Steel (Material A)</td>
<td>12 mm</td>
<td>0.13</td>
</tr>
<tr>
<td>NK-B Grade Commercial Steel (Material B)</td>
<td>12 mm</td>
<td>0.12</td>
</tr>
<tr>
<td>SS-2</td>
<td>12 mm</td>
<td>0.11</td>
</tr>
<tr>
<td>SS-4</td>
<td>11 mm</td>
<td>0.17</td>
</tr>
</tbody>
</table>

### Table 9. Critical Length of Defect (Corresponding to 2C in Fig. 12)

<table>
<thead>
<tr>
<th>Temperature (°C)</th>
<th>2C crit (mm)</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>10 kgf/mm²</td>
</tr>
<tr>
<td>0°C</td>
<td>20 mm</td>
</tr>
<tr>
<td>10°C</td>
<td>44 mm</td>
</tr>
</tbody>
</table>

Test Temperature (°C)

![Fracture Toughness of SS-2 and Commercial Steel of NK-A Grade](image)

![Model of Defect in Infinite Plate](image)
Fatigue test

In order to discuss on the potential damages caused by connecting old plate and new ones by welding, also in order to investigate into the effects of corrosion in old plates on the fatigue strength of them, fatigue tests were conducted by using specimens from old plates DP-1 (deck plate). Fig. 13 shows the configuration of specimen used. A new weldable mild steel SM41A plate and an old plate were connected by welding for one type of specimen ("A" type, 10 specimens of this type were tested). For the other type, two new SM41A plates were welded ("B" type, 6 specimens of this type were tested). Single-V groove, 3 passes from one side and 2 passes from the other side were used for welding. The welding conditions were: diameter of electrode = 4 mm, voltage = 20-25 V, current = 150-180 A and travelling speed = 133 mm/sec. Plate thickness of a new plate and welding conditions were similar to those used at repair works of K-maru.

![Fatigue Test Specimen](image)

**Fig. 13 Fatigue Test Specimen**

Figure 14 shows the fatigue life diagram expressed by a load and a nominal stress. Figure 15 shows the fatigue life diagram expressed by a stress at the section with a minimum sectional area. Number C-1, C-2 etc. in Figures 14 and 15 are test specimen numbers. If a welding is sound enough, fatigue cracks in a butt weld joint specimen of SM41A initiate at weld toe and propagate to the thickness direction. But in the "B" type specimens tested, because of relatively large insufficient penetration of weld metal, fatigue cracks initiated at weld defects and propagated in weld metal to final failure. In spite of the fact that the same welding was used for "A" type specimen, fatigue cracks initiated at several points simultaneously on a corroded surface of a old plate and propagated to a thickness direction of a plate to final failure, with one exception (specimen C-4). It could be observed from macro fracture surfaces that many fatigue cracks initiated at many parts in a rough corroded surface and they merged gradually into one and proceeded to a final failure. Figures 16, 17 and 18 show the enlarged appearances of corroded surface of same test specimens shown in Figures 14 and 15. One side of the plate DP-1 (deck) was, at the time of usage as a part of a hull of K-maru, exposed directly to inside of the ship and so protected by many accumulated paint layers. This side of the plate was not corroded and kept a flat surface. But the other side of the plate was under a wooden deck and not protected against corrosion. And so, part of this side were excessively corroded.

In specimen C-3 (Fig. 16), plate thickness decreases gradually to its longitudinal direction. In C-4 (Fig. 17), decrease in thickness is very large and there is even a corrosion hole. In C-6 (Fig. 18), plate thickness shows abrupt changes. In case of C-4, a crack initiated at the edge of the hole played as one of cracks initiated at many points on a corroded surface.

From these results and discussions, we estimates that the reduction in fatigue strength of corroded plates is due to the combined effects of reduction in plate thickness and surface roughness produced by corrosion. From the fact that cracks initiate at corroded surface and propagate in corroded plates even in case of a poor welded joint specimen, the reduction in strength due to plate thickness decrease and rough surface caused by corrosion is more significant from the navigation safety of K-maru than the effects of welding of rimmed steel plates on the strength.

**DISCUSSION ON NAVIGATION SAFETY**

**Basic concept on ship hull strength**

There are several view points of strength of structures. One view is to assume that individual member consisting a structure works additively to the total strength of the structure (hereafter this view is called "static strength"). Another view is to assume that a crack initiates at the weakest part in a structure and the crack propagates through each structural member independently to other members to final failure (typical case of this is the brittle fracture strength). Other than these, there is a view of strength against emergencies such as collisions or groundings as in cases of ships.

**Static strength**

Static strength of a ship as a whole can be discussed based on her I/Y value and buckling strength. From the measurement of deterioration due to corrosion at midship of K-maru, it is considered that the reduction of I/Y due to deterioration is nearly 5% assuming that there is negligibly small amount of shift of the neutral axis. But there are possibilities of reduction in local buckling strength due to local severe corrosion and deformations.

**Brittle fracture**

There remains a possibility that there, in K-maru, are included such plates with low fracture toughness as the old plate...
SS-2, and if repairs of the plates by welding are continued to do, it is possible that brittle fracture might initiate from fatigue cracks from corroded surface or from those due to welding not found at the time of inspections. The latter cracks are invited by use of welding which is not anticipated at the time of the construction of K-maru.

"Navigation safety"

When it comes to discuss the navigation safety or seaworthiness of K-maru, not only static and brittle fracture strength, but also strength distribution effects of excessively corroded parts and holes in plates of hull on the strength of the ship, and also the safety at emergencies are to be taken into consideration.

The strength distribution can be expressed by using $S/P$. Where $S$ is the strength of individual member of a structure and it is a decreasing function of time. $P$ is the external force acting on members and it is independent of time in case that the ship keeps the same navigation conditions over her life. Plate thickness reduction and surface roughness due to corrosion, generations of cracks and deformations of individual member should be considered for the evaluation of $S$. Tensile, compressive and repeating loads are to be adopted for $P$. Figure 19 shows the conceptional view of $S/P$ distribution. As time passes, the $S/P$ distribution around $M_e$ (solid line) at initial stage of a ship shifts to the distribution around $M_r$ (broken line). In this concern, K-maru is considered at the stage of so called wear out failure in the bath tub curve of ship age to failure pattern diagram (Fig. 20). And even if she were carefully repaired, the $S/P$ distribution described in Fig. 19 continues to shift apart from $M_e$, and the peak value of the distribution also continues to decrease.

"CONCLUSION"

Article 1 of Ship Safety Law (Japan) says "ships of Japan should not be used for navigation if their seaworthiness is not guaranteed and not equipped with facilities necessary for keeping safety of lives stipulated by this law". It is considered that the seaworthiness of the ship is guaranteed by such facilities stipulated by Article 2 of...
this law as hull, machinery, life saving, living and electric ones. K-\textit{maru} which is equipped with those facilities as mentioned above, and there seems to be nothing to be considered for safety as far as the law is concerned. But it is questionable, however, that the law can be applicable directly to such a long life ship as K-\textit{maru}. Steel plates, used at the time of construction of K-\textit{maru}, were rimmed ones and they were connected by rivets. But today, they are changed to killed steel plates and connections by welding. These changes give great influences to the seaworthiness of K-\textit{maru} which has been used for so long time by repairing so many times and places. Therefore we discussed the safety of K-\textit{maru} not based on Ship Safety Law but on the theories and experiences established today, and concluded that the safety of K-\textit{maru} in future could not be guaranteed.

Other than articles concerning seaworthiness of the ship, Ship Safety Law has Article 13, which says "if more than 10 crew members in a ship appeal the serious defects with reference to seaworthiness, living and other life saving facilities referring to stipulations in this law, the government office concerned should make the necessary investigations and, if necessary, give punishments stipulated by Paragraph 3, Article 12 (suspension of navigation and other punishments)". From this viewpoint, vacant holes due to corrosion, which may give big psychological effects to crew on the safety of their ship, should be considered more seriously than it is estimated theoretically.

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REFERENCES

6) For example, S. Abe, "Hull Damages and Maintenance Measures of NK Registered Ships", Transactions of Nippon Kajii Kyokai, No. 175, PP. 18-32, 1981.